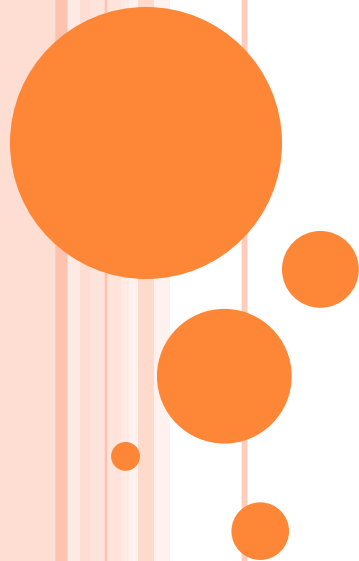


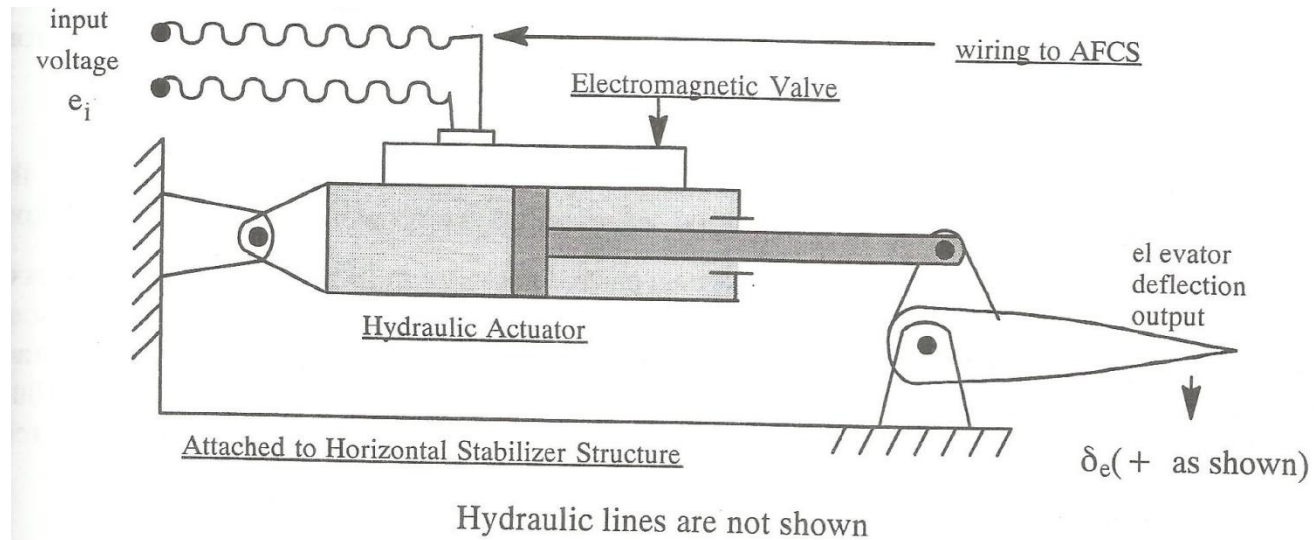
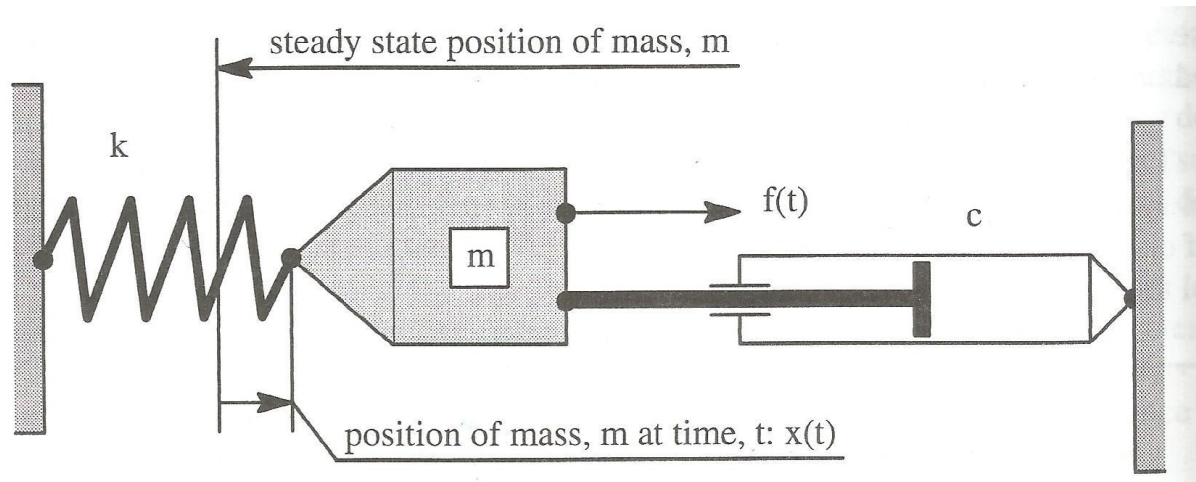
BASIC FEEDBACK AND ITS VARIOUS TYPES



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- The fundamental idea behind feedback control is to **modify the stability characteristic** of given system which has **unsatisfactory inherent stability behavior**.
- For example if a system has insufficient damping a feedback may arranged in order to improve the damping.
- As a general rule, any one of the following feedback may be used to improve the damping
 - **Position feedback (also called stiffness feedback)**
 - **Velocity feedback (also called rate feedback)**
 - **Acceleration feedback**





Example of Spring – Mass – Damper system

The equation of motion for the system

$$m\ddot{x} + c\dot{x} + kx = f(t)$$

Taking Laplace transform for the initial condition yields

$$ms^2 x(s) + csx(s) + kx(s) = f(s)$$

$$(ms^2 + cs + k)x(s) = f(s)$$

$$\frac{x(s)}{f(s)} = \left(\frac{1}{ms^2 + cs + k} \right)$$

The dynamic stability behaviour of the system is determined completely by the roots of its characteristic equation also called "system equation".

$$ms^2 + cs + k = 0$$

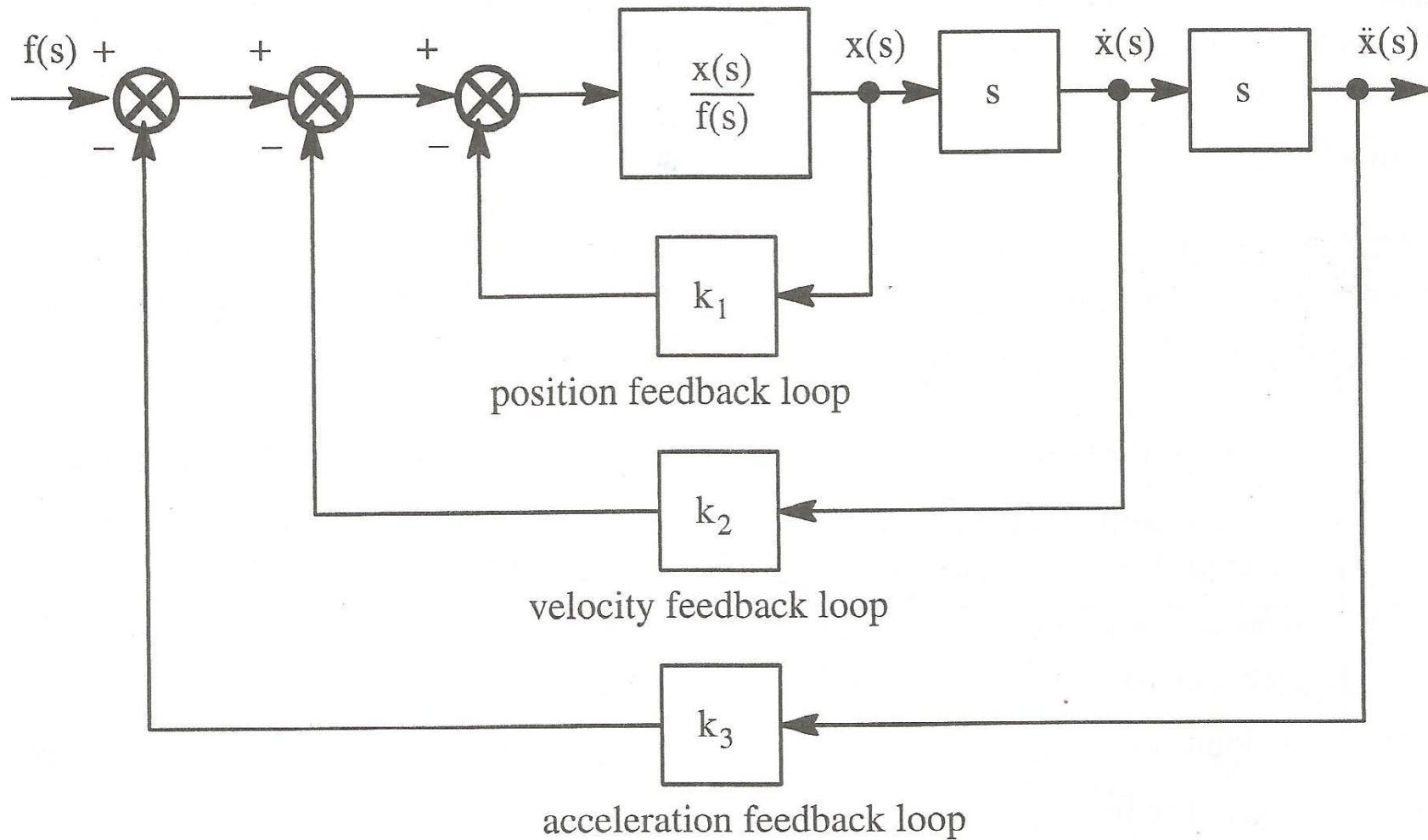


The roots of this equation taking the following form

$$s_{1,2} = \frac{-c \pm \sqrt{c^2 - 4mk}}{2m}$$

Note : The system will be stable as long as $C > 0$.





Block diagram of the system with three feedback loops



$$m\ddot{x} + c\dot{x} + kx = f(t)$$

The above equation of motion now modified as (because of feedback)

$$m\ddot{x} + c\dot{x} + kx = f(t) - k_1x - k_2\dot{x} - k_3\ddot{x}$$

Taking the laplase transformation for zero initial condition

$$\{(m + k_3)s^2 + (c + k_2)s + (k + k_1)\} x(s) = f(s)$$

$$\frac{x(s)}{f(s)} = \left(\frac{1}{(m + k_3)s^2 + (c + k_2)s + (k + k_1)} \right)$$

The characteristic equation for the feedback system is

$$(m + k_3)s^2 + (c + k_2)s + (k + k_1) = 0$$



The roots of this characteristic equation may be expressed as

$$s_{1,2} = \frac{-(c + k_2) \pm \sqrt{(c + k_2)^2 - 4(m + k_3)(k + k_1)}}{2(m + k_3)}$$

- The above equation is said to be “**augmented system equation**”. From the above equation it is clear that the role of the feedback gain k_1, k_2, k_3 is to alter the root in the S-plane.
- The effect of varying the **position, velocity and acceleration feedback gains** are shown in the figure.



Root - loci for : $m\ddot{x} + c\dot{x} + kx = f(t) - k_1x - k_2\dot{x} - k_3\ddot{x}$

For open loop system:

$$m = 5 \text{ slugs}$$

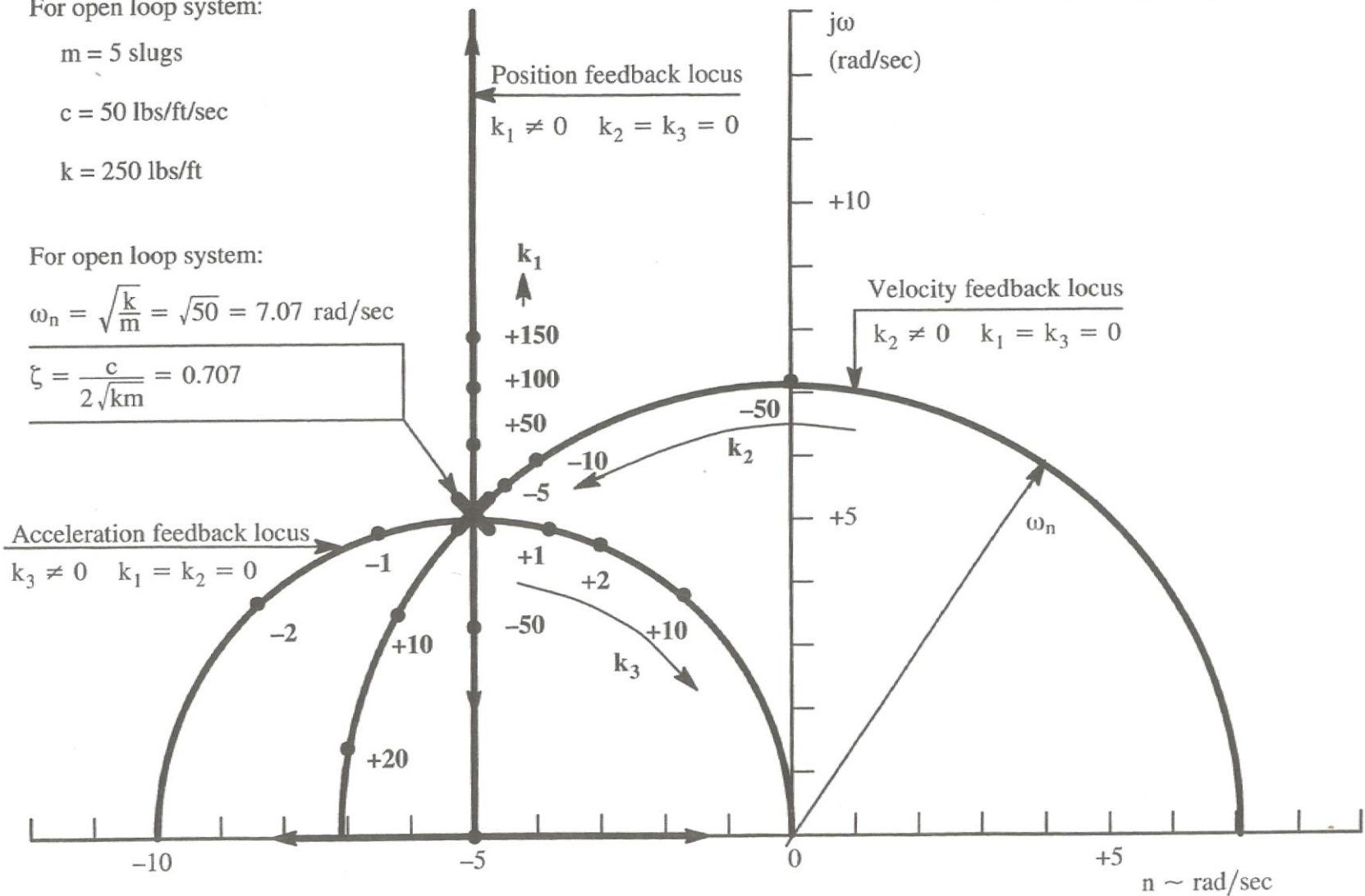
$$c = 50 \text{ lbs/ft/sec}$$

$$k = 250 \text{ lbs/ft}$$

For open loop system:

$$\omega_n = \sqrt{\frac{k}{m}} = \sqrt{50} = 7.07 \text{ rad/sec}$$

$$\zeta = \frac{c}{2\sqrt{km}} = 0.707$$



Root locus for Position–Velocity and Acceleration feedback

POSITION FEEDBACK

- Position feedback k_1 affects both the undamped natural frequency and damping ratio of the closed loop system.
- The root locus of the case is a straight line which passes through the open loop system poles.
- It is noted that the real part of the closed loop system roots remains constant as k_1 is varied while k_2 and k_3 are kept at zero.



VELOCITY FEEDBACK

- Velocity feedback or rate feedback k_2 affects the damping ratio of the closed loop system only.
- The root locus in the case is a circular around the origin of the S-plane.
- It is noted that the undamped natural frequency of the closed loop system remains constant as k_2 is varied while k_1 and k_3 are kept at zero.

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ACCELERATION FEEDBACK

- Acceleration feedback k_3 affects both the undamped natural frequency and damping ratio of the closed loop system.
- The root locus is a circle with the origin at the point $\omega_n = -0.5 \text{ rad/sec}$.
- Note that k_3 is varied towards infinity, the undamped natural frequency tends towards zero.



- By selecting any combination value of k_1, k_2, k_3 it is possible to place the poles of the closed loop system at an arbitrary desired location in the S-plane



THANK YOU

